

Modeling and Analysis of a Direct Expansion Geothermal Heat Pump (Dx): Part II-Modeling of Water-Refrigerant Exchanger

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Introduction: In this section, we simulate the heat exchanger system in one dimension characterized by two coaxial tubes (Figure 2) with ribbed inner tube using the equations of conservation of mass, conservation of momentum and energy.

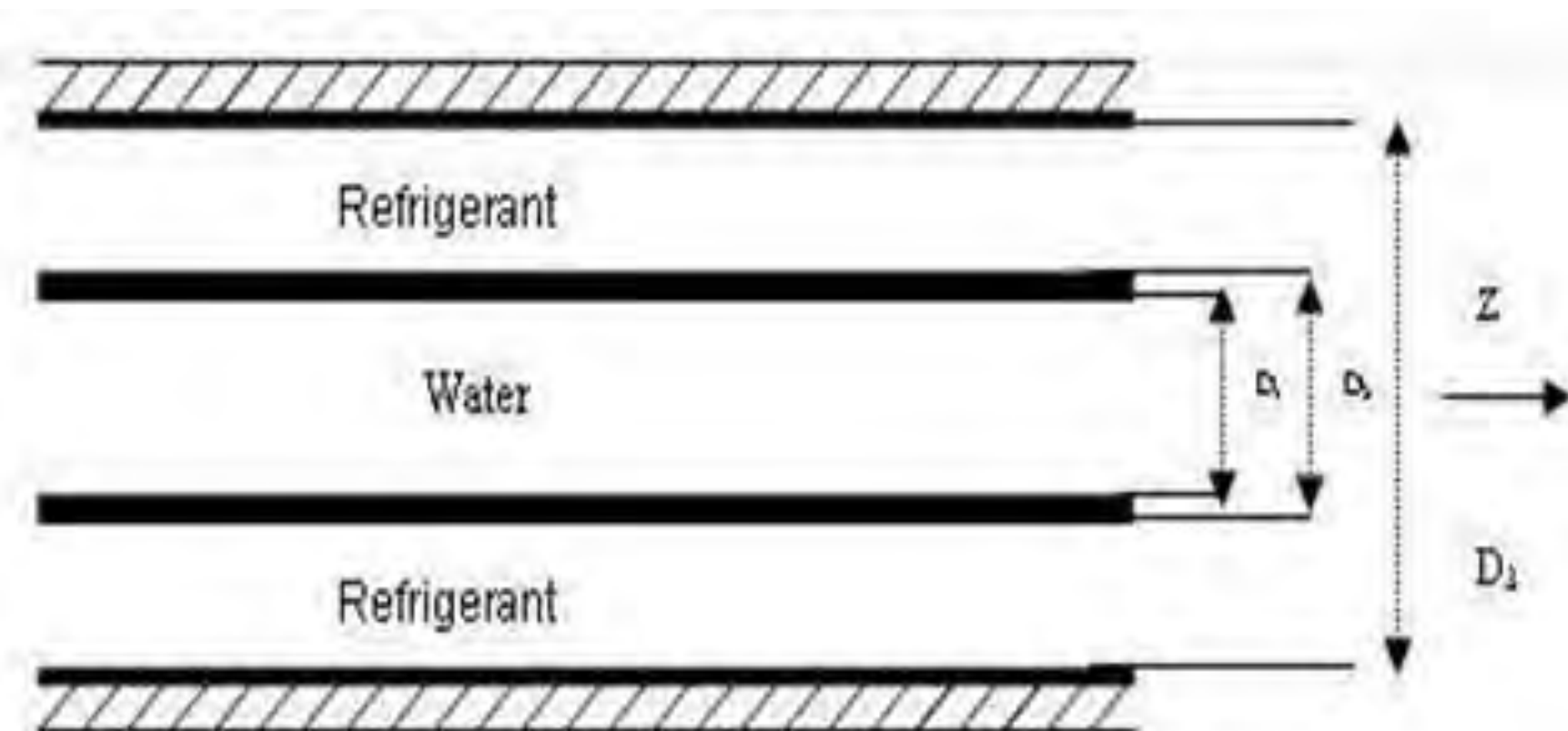


Figure 1. heat exchanger model



Figure 2. Exchanger diagram

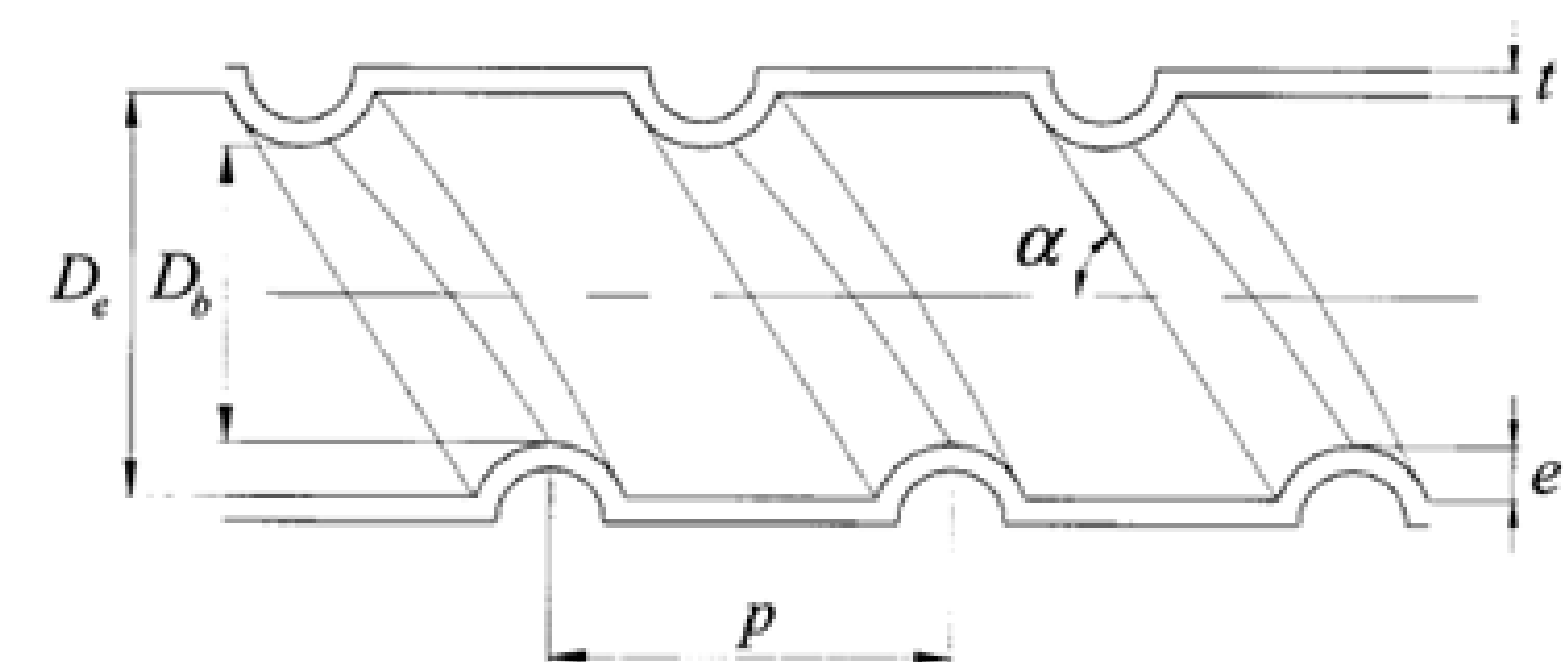


Figure 3 . Exchanger geometry

Computational Methods:

The COMSOL PDE interface and two heat transfer interface (solid, fluid) are used to solve the governing equations:

Refrigerant(R22)

$$S_f \frac{\partial \rho}{\partial t} + \frac{\partial \dot{m}}{\partial z} = 0$$

$$S_f \frac{\partial \dot{m}}{\partial t} + \frac{2\dot{m}\partial\dot{m}}{\rho \partial z} - \frac{\dot{m}^2 \partial \rho}{\rho^2 \partial z} + S_f \frac{\partial P}{\partial z} = -S_f^2 F_{vol}$$

$$S_f \rho \frac{\partial h}{\partial t} + \dot{m} \frac{\partial h}{\partial z} - S_f \frac{\partial P}{\partial t} - \frac{\dot{m} \partial P}{\rho \partial z} = S_f Q_v$$

$$h = (1 - x)h_f + xh_g$$

$$\rho = (1 - \alpha_b)\rho_f + \alpha_b\rho_g$$

Inner wall

$$\rho_p C_p \frac{\partial T_p}{\partial t} - k_p \frac{\partial^2 T_p}{\partial z^2} = -Q_{ep} + Q_{rp}$$

Water

$$\rho_e \left(\frac{\partial T_e}{\partial t} + v \frac{\partial T_e}{\partial z} \right) = \frac{4D_{1p} h_{ep}}{D_{1A}^2} (T_p - T_e)$$

Results: We present the results of the heat exchanger in the heating mode when it acts as a condenser.

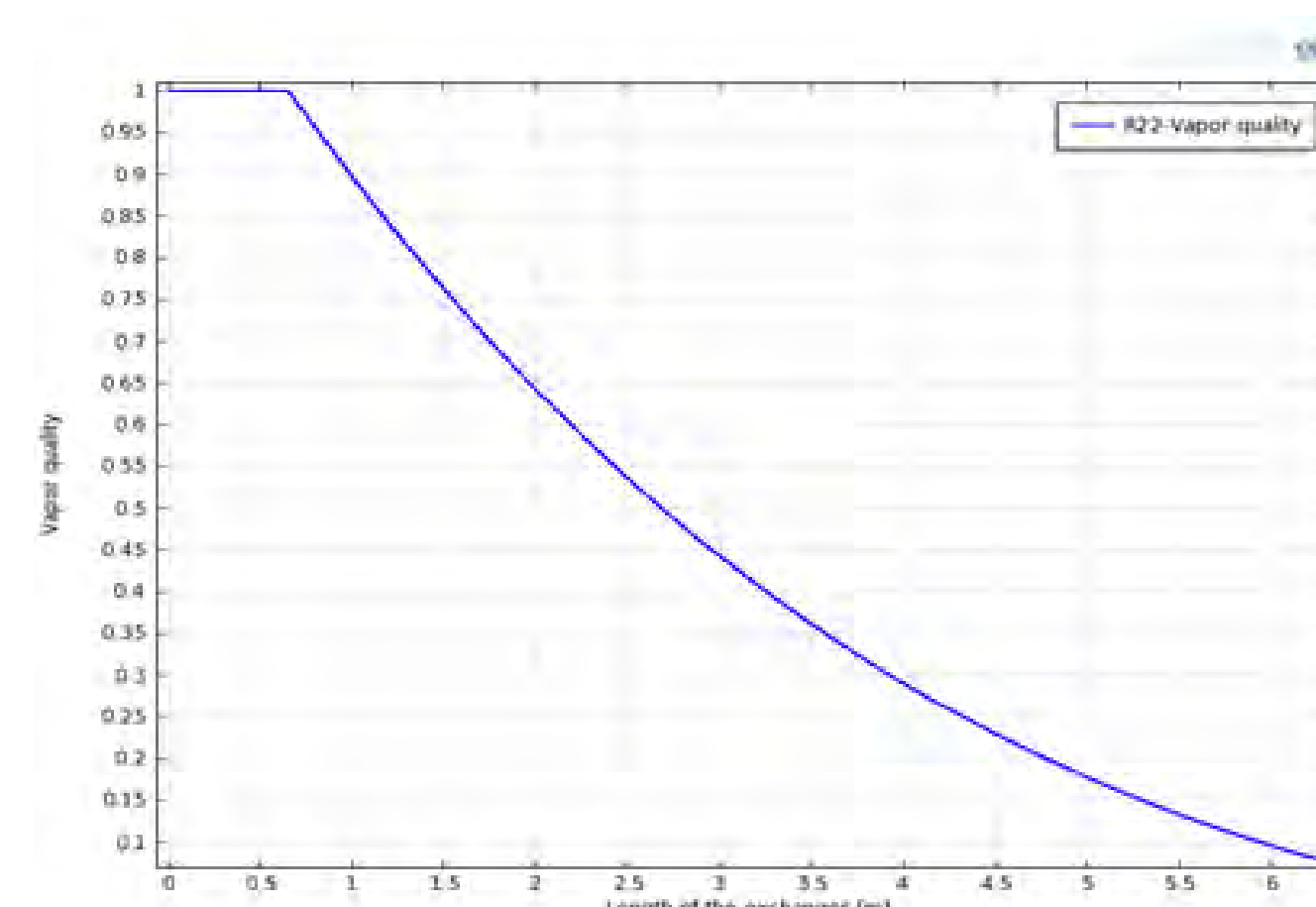


Figure 4. Vapor quality

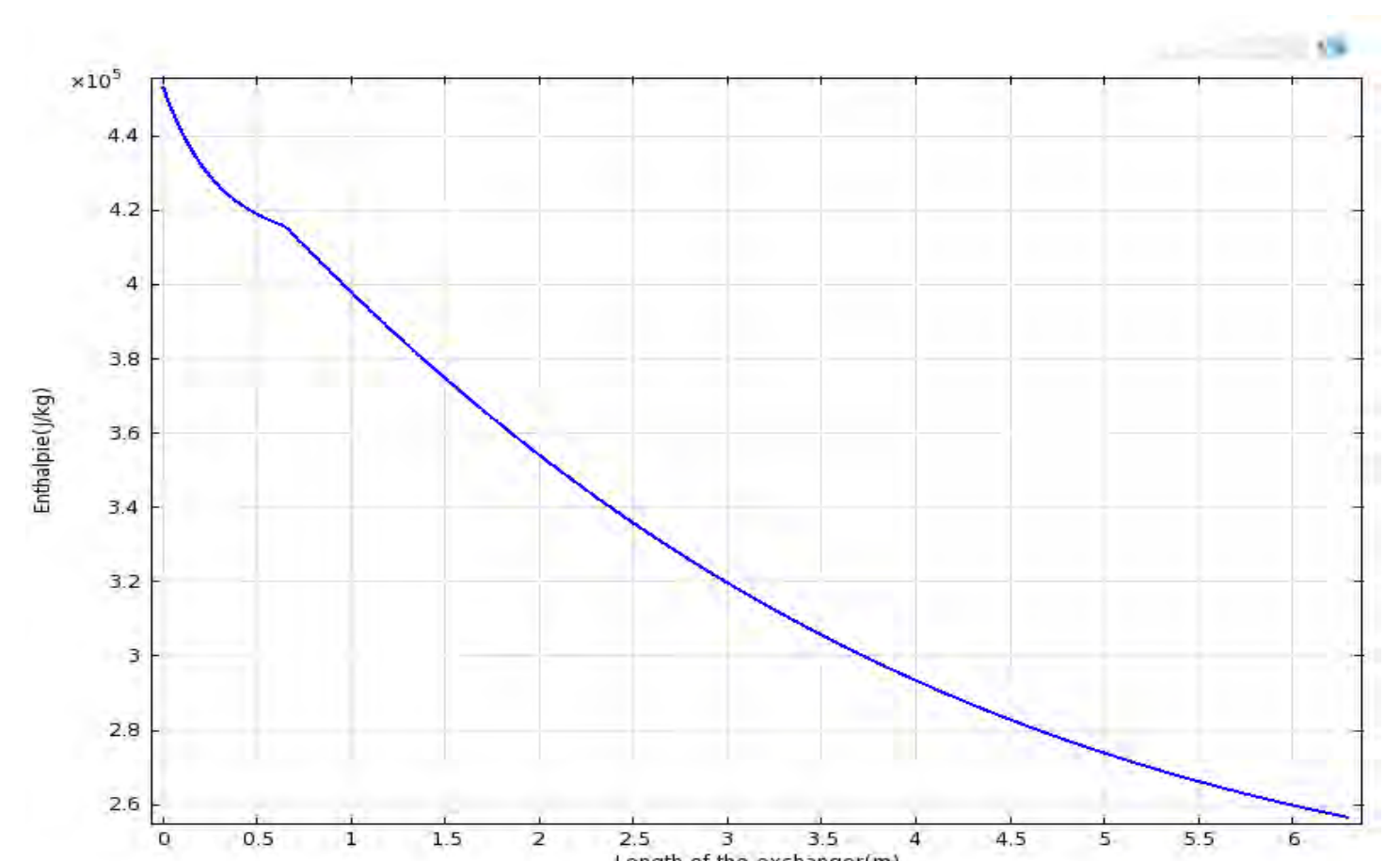


Figure 5. Enthalpy

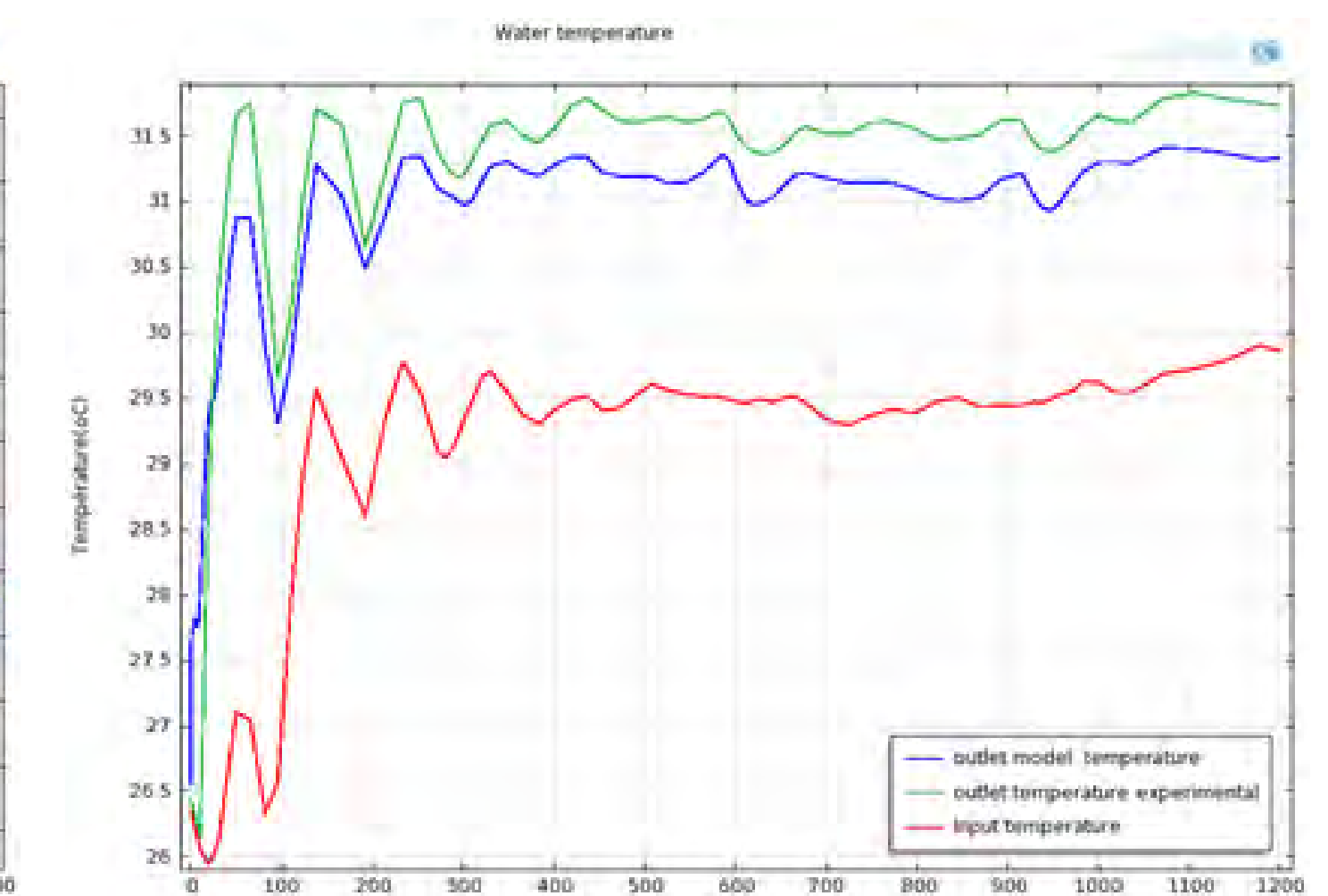
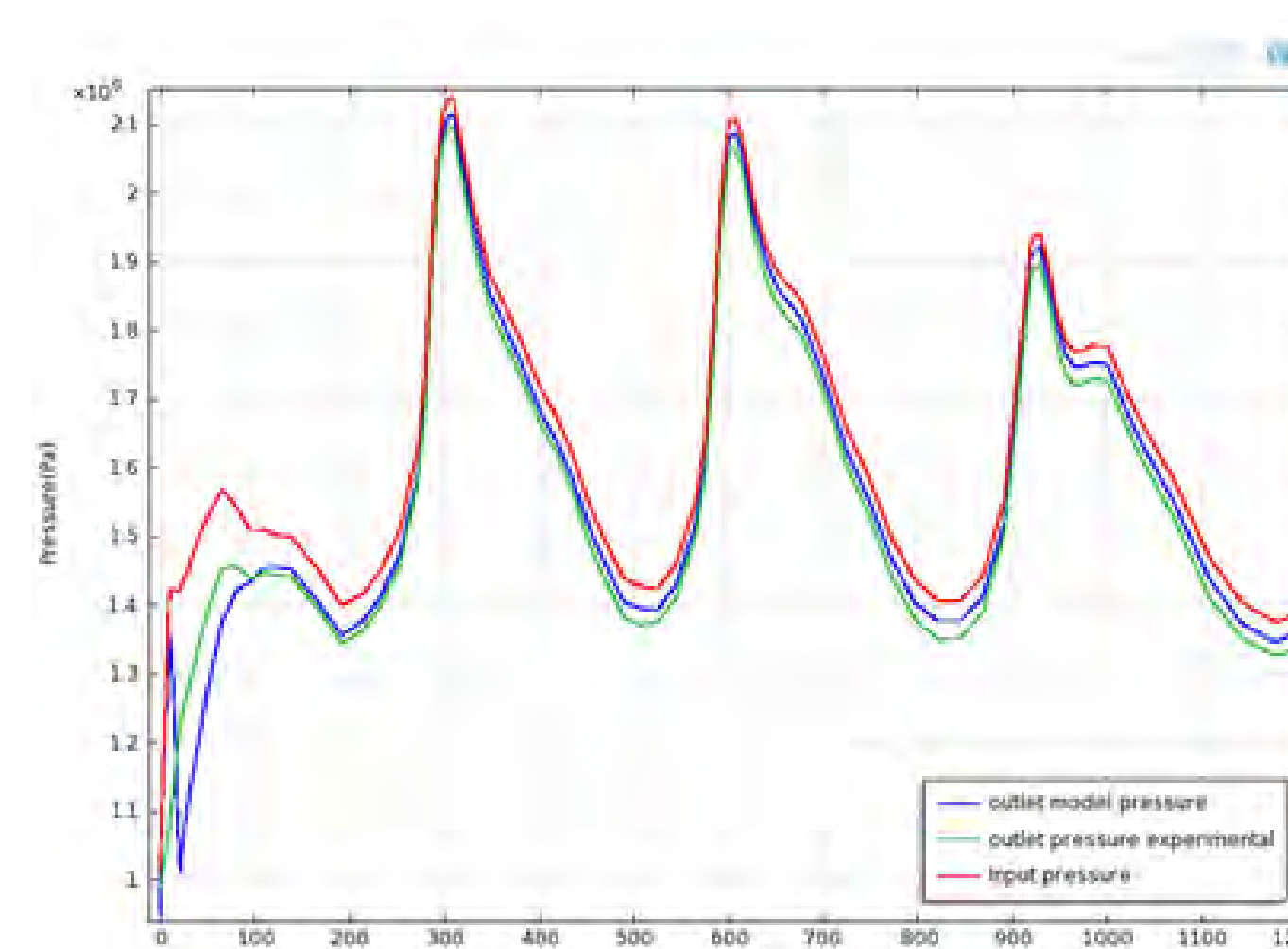


Figure 6. Experimental validation

Conclusions: The numerical model of water-refrigerant heat exchanger with Comsol.

The validated numeric model obtained will be coupled with the ground heat exchanger for developing the global numerical model of direct expansion geothermal heat pump.

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