

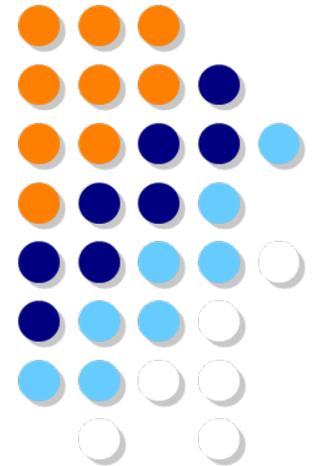
Application of Solution Mapping to Reduce Computational Time in Actively Cooled Power Electronics

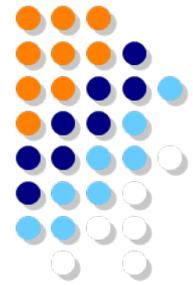


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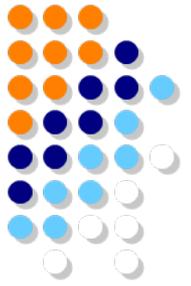
October 9, 2008





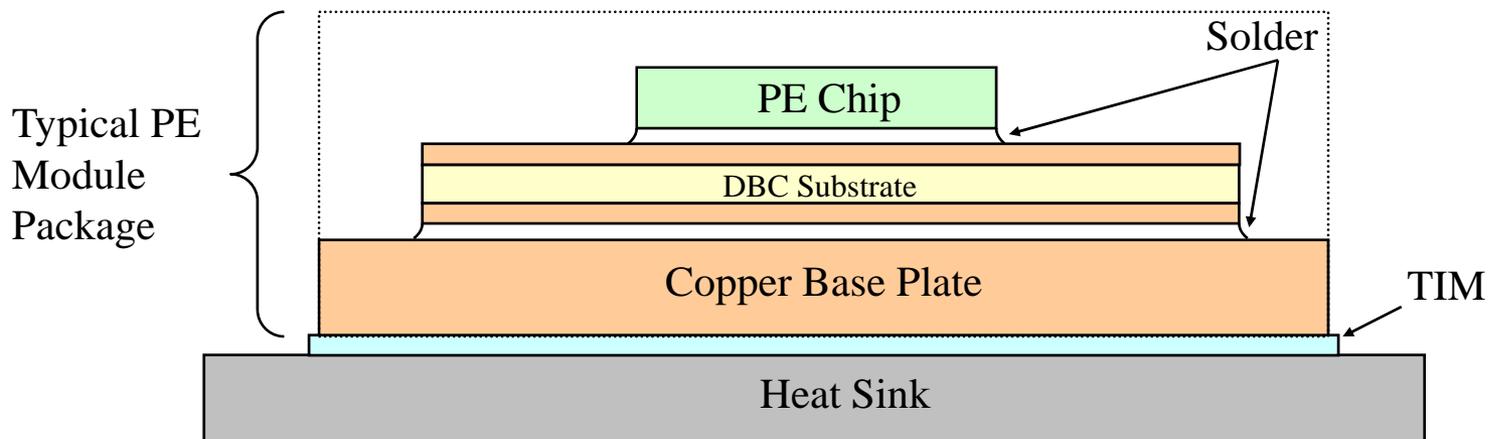
Acknowledgements

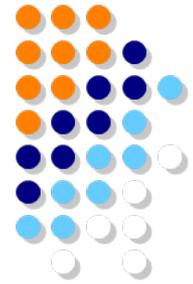
- Project Sponsored by:
Oak Ridge National Laboratory
- Work in support of Direct-Cooled Power Electronics Substrate Project funded by DOE FreedomCar Program



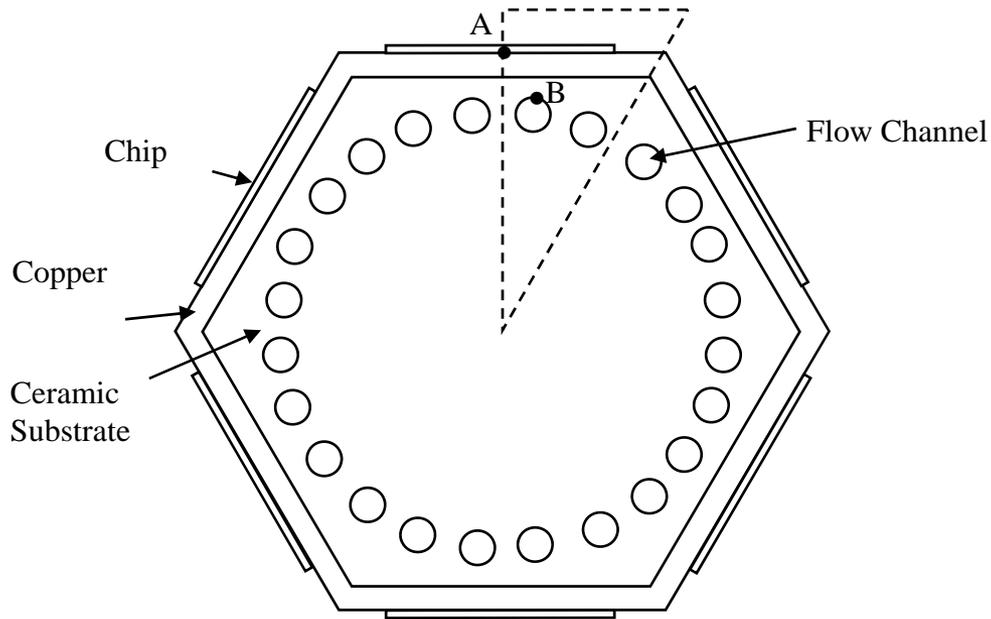
Problem Description

- Typical packaging has many layers
- Thermal resistance is large
- Commercial applications can require a coolant temperature over 100° C
- Thermal resistance must be reduced

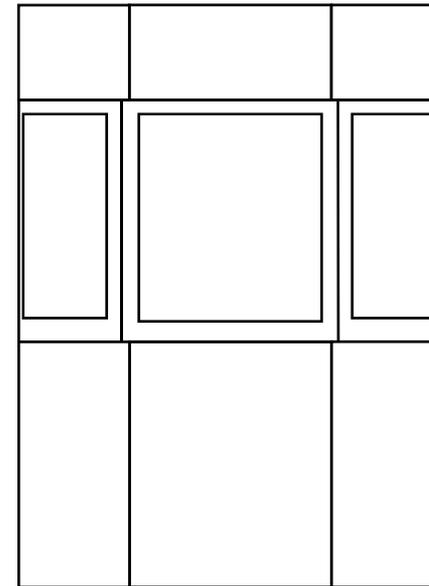




Proposed Solution

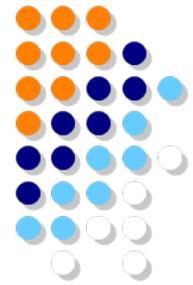


Front View



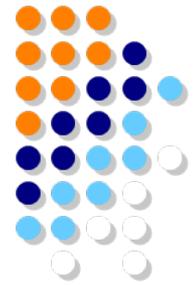
Top View

Provision Patent 61/037,129



Proposed Solution (ii)

- Embeds heat sink into ceramic
- Eliminates TIM, copper base plate, one solder layer, and aluminum heat sink
- Thermal performance of coolant channel design is modeled to compare to design limitations.



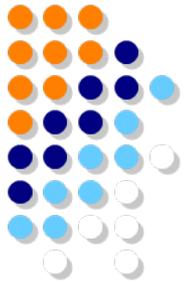
Model Setup

- Fluid Dynamics
 - Incompressible Navier-Stokes
 - Continuity
- Heat Transfer
 - Conduction and Convection
- Constant Properties
- Steady State to predict worst-case-scenario
- Maximum Temperatures
 - Fluid – 130° C
 - Interface – 150° C

$$\rho \bar{u} \cdot \nabla \bar{u} = \nabla \cdot \left[-p\mathbf{I} + \eta(\nabla \bar{u} + (\nabla \bar{u})^T) \right]$$

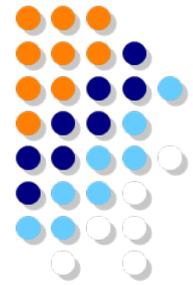
$$\nabla \cdot \bar{u} = 0$$

$$\rho C_p \bar{u} \cdot \vec{\nabla} T = Q''' + k \nabla^2 T$$



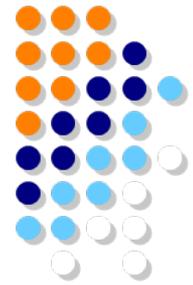
Solution Strategy

- Solve 2-D axisymmetric flow field
- Map solution to 3-D cylinders
(Extrusion Coupling Variables)
- Apply thermal boundary conditions
- Solve for temperature distribution



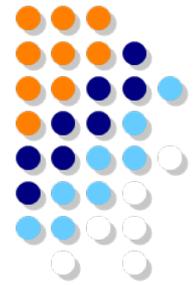
Boundary Conditions

- Inlet Velocity, $Re_{D,in}$
- Outlet, Pressure, no viscous stress, $p_0=0$
- Walls, no slip
- Chip Heat Load, $1.78e9 \text{ W/m}^3$
- Inlet Temperature, 105° C
- All other boundaries thermally insulated

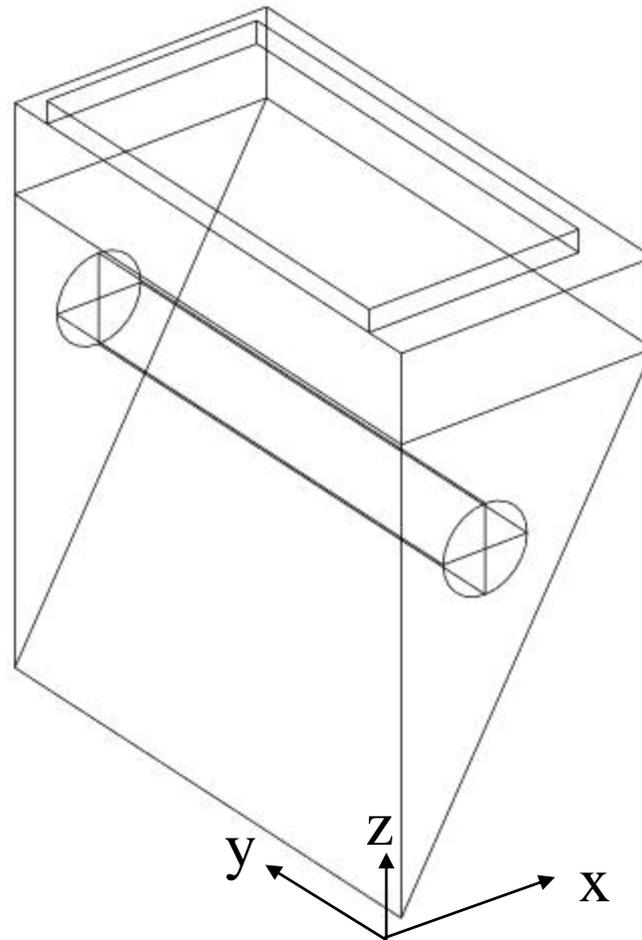


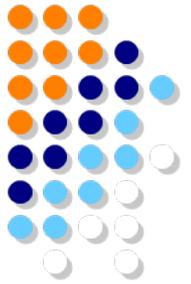
Solution Mapping

- Useful for simple flow field in more complex structure
- Solve 2-D axisymmetric flow field
- Map solution to 3-D cylinder using Extrusion Coupling Variables
- Translate axis as necessary
- Separate variables into directional components
- Incorporate into convection heat transfer
- Solve temperature distribution



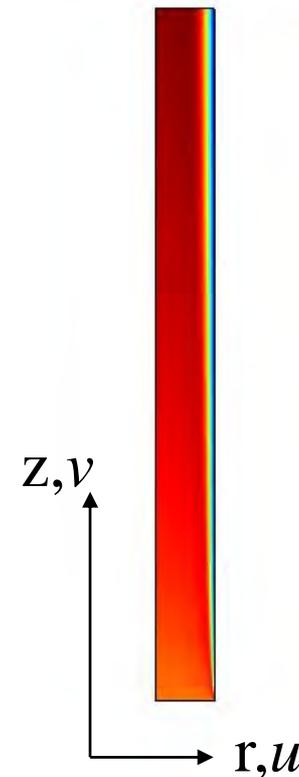
Solution Mapping (i)

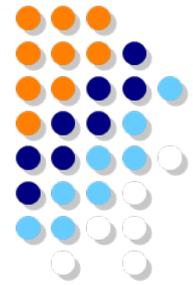




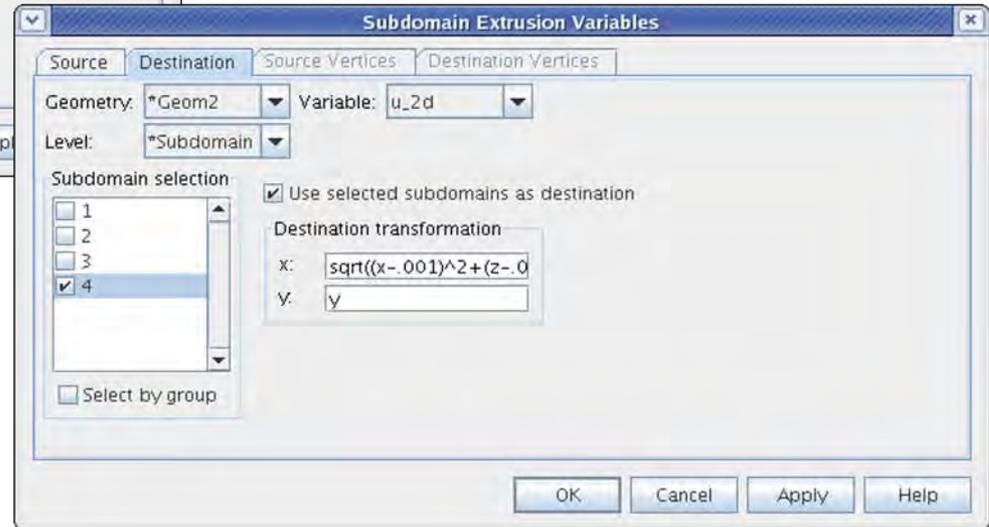
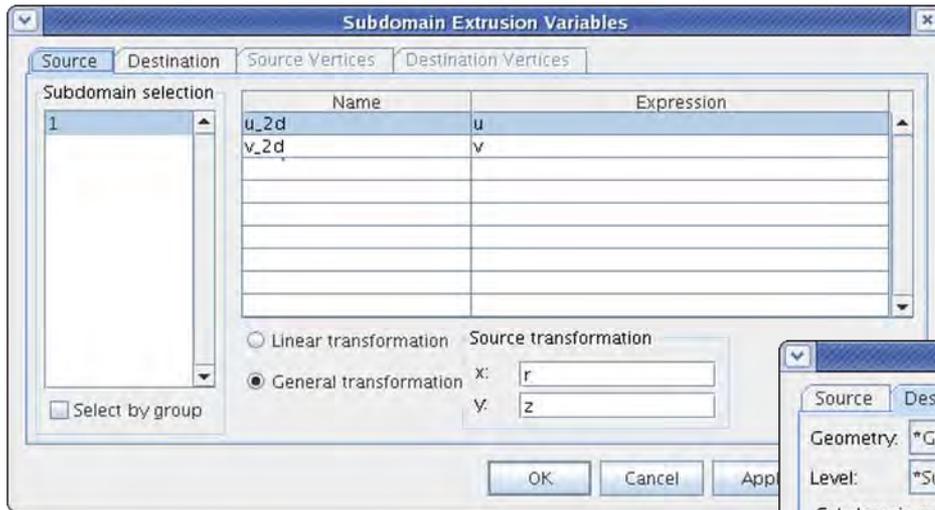
Solution Mapping (iii)

- 2-D axisymmetric solution
- Solved parametrically to desired input velocity or Reynolds number



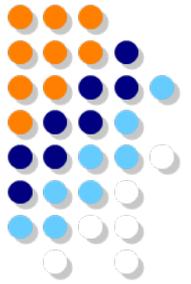


Solution Mapping (iv)

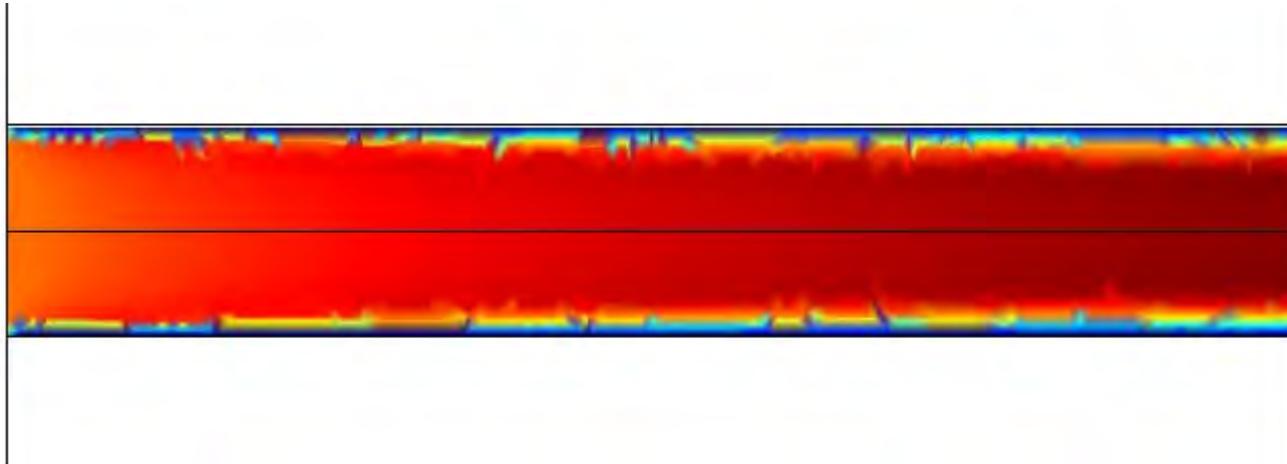


$$\sqrt{(x - x_i)^2 + (z - z_i)^2} = r$$

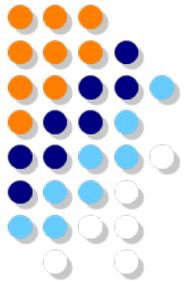
$$y = z$$



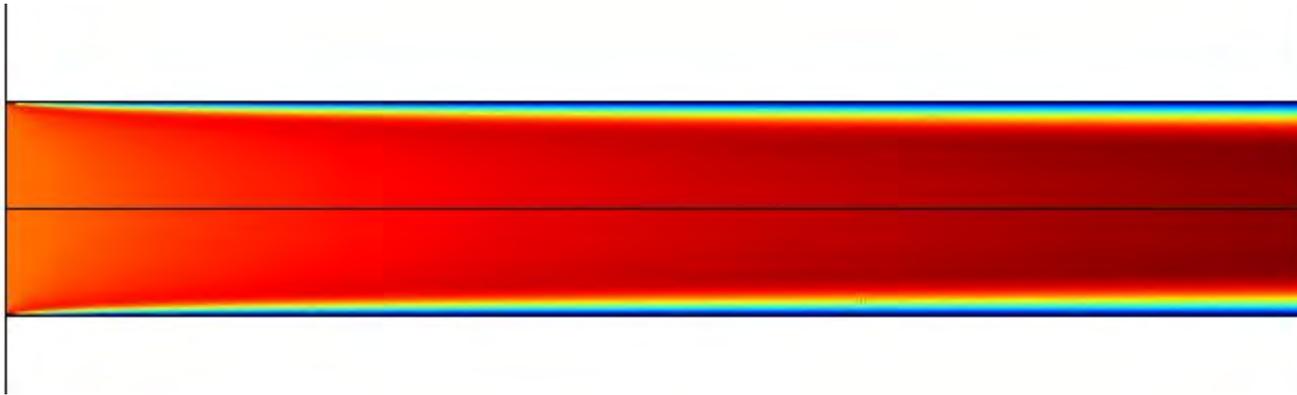
Solution Mapping (v)



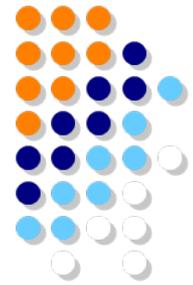
- Initial model update uses initialized (coarse) mesh
- Mesh refinement is necessary to transfer velocity data to convection regime



Solution Mapping (vi)



- Fine mesh better resembles the accuracy of the 2-D solution



Solution Mapping (vii)

Subdomain Expressions

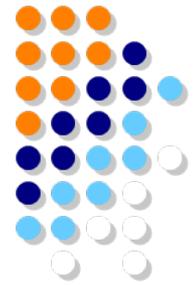
Subdomain selection

1
2
3
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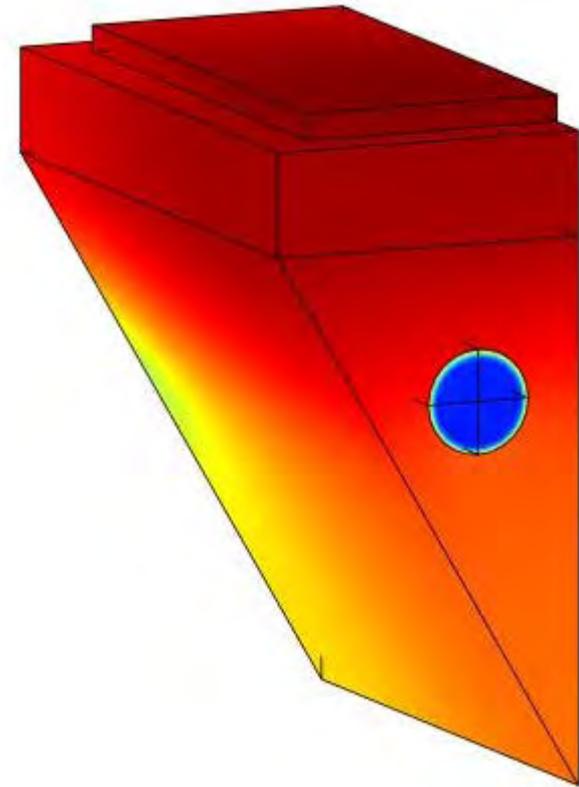
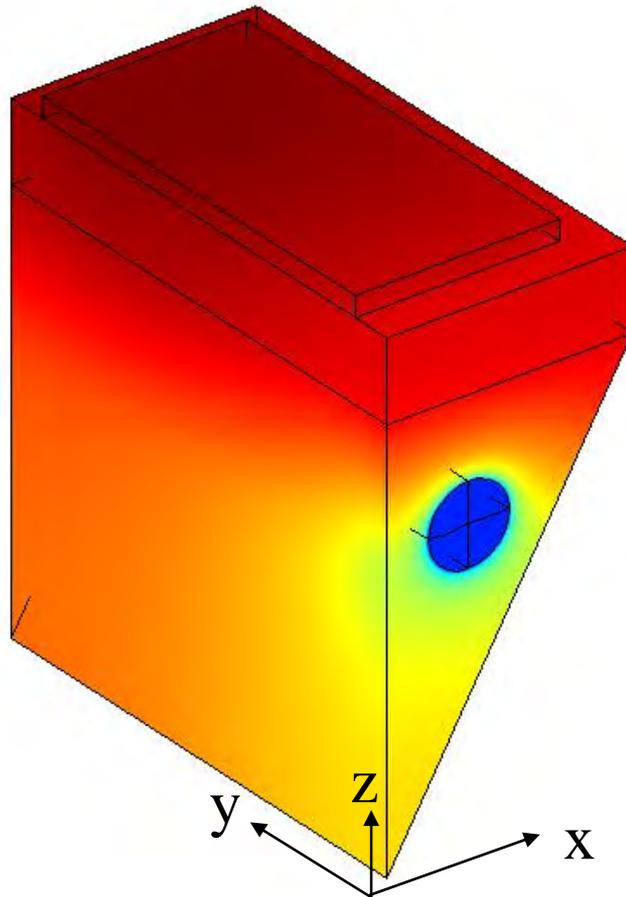
Select by group

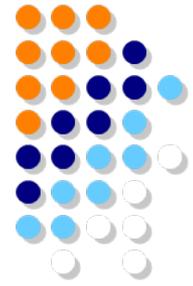
Name	Expression	Unit
x2	$-0.002 + x$	m
z2	$-0.0015 + z$	m
theta	$\text{atan}(z2/x2)$	rad
u2	$u_2d \cdot \cos(\text{theta})$	[]
w2	$u_2d \cdot \sin(\text{theta})$	[]

OK Cancel Apply Help



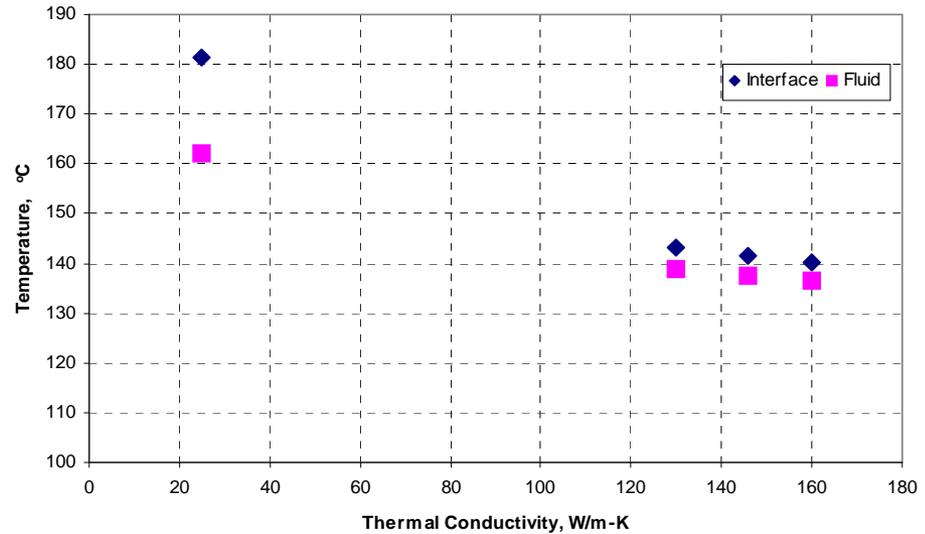
Solution Mapping (viii)



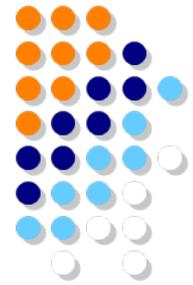


Results

- High thermal conductivity ceramics produce lower maximum interface and fluid temperatures
- Chip temperatures are within design limits
- Working Fluid is above its boiling point

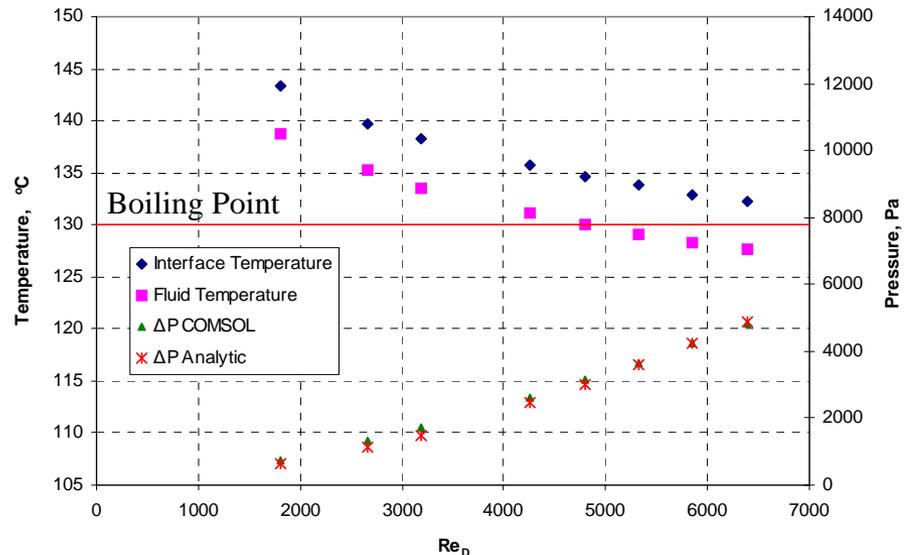


Maximum Interface and Fluid Temperatures of the Four Ceramic Materials at $Re_{D,in}$

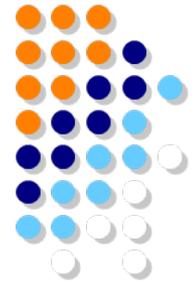


Results (ii)

- Increasing inlet Re_D , decreases maximum temperatures
- Pressure drop is small
- Model pressure drop coincides well with analytic calculations

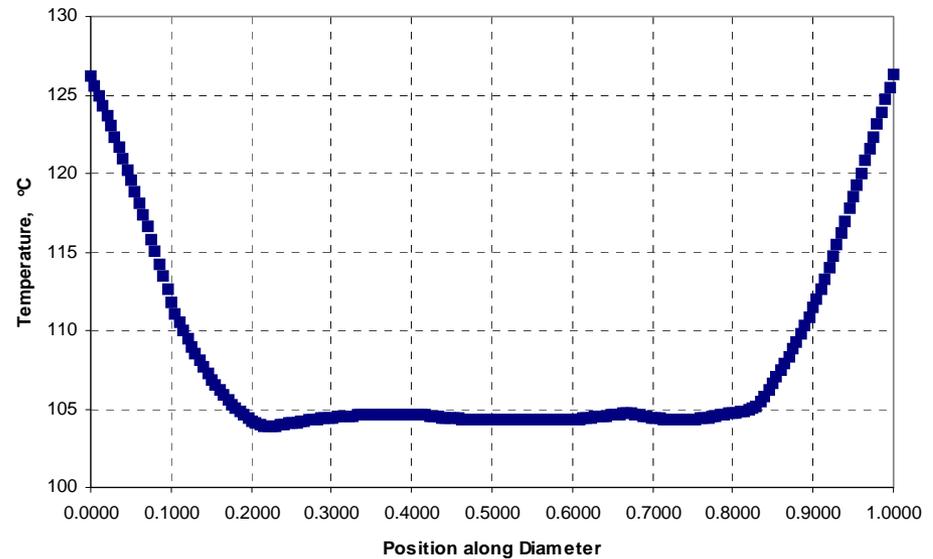


Variations of Maximum Interface and Fluid Temperatures and the Pressure for Ceramic 4

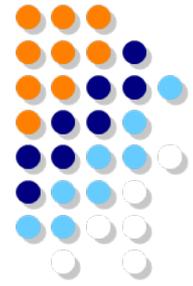


Results (iii)

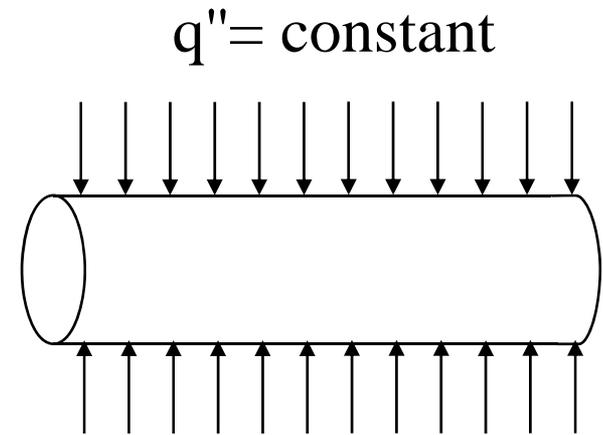
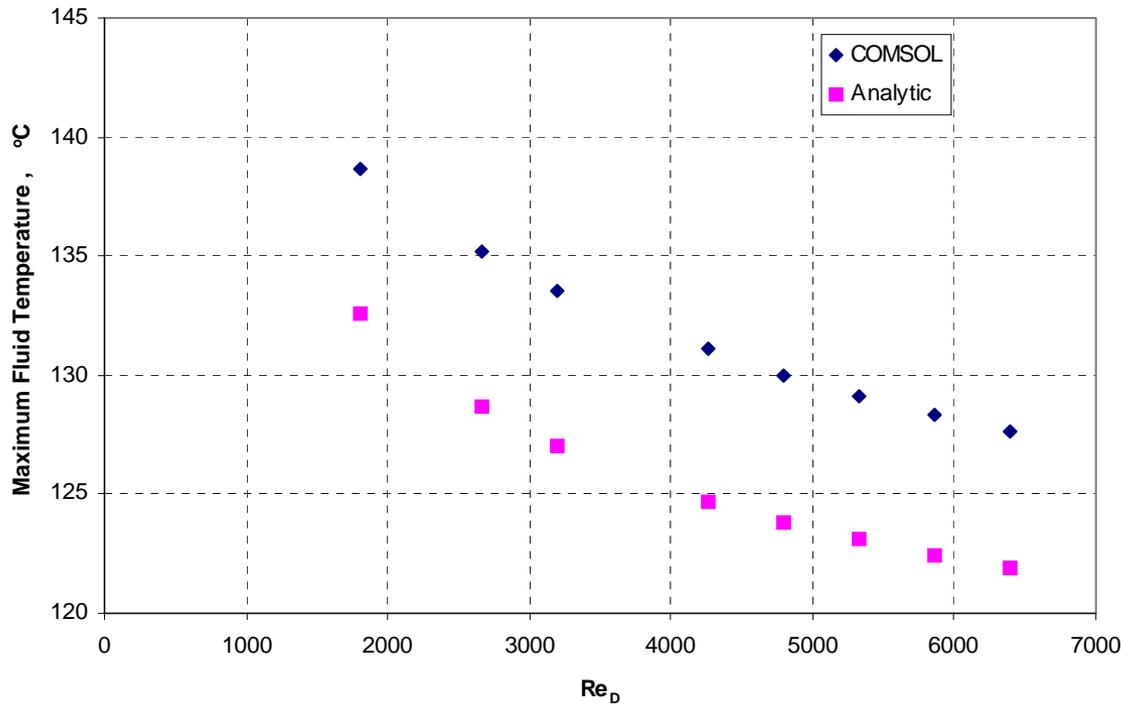
- Large core of “cold” fluid at channel exit (41.3%)
- Diameter at mass manufacturing limit
- To improve cooling
 - Surface enhancement
 - Thermal conductivity enhancement



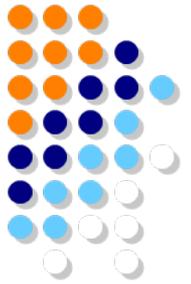
Fluid Temperature Along Diameter at Outlet of Hottest Channel in Ceramic 4



Solution Accuracy

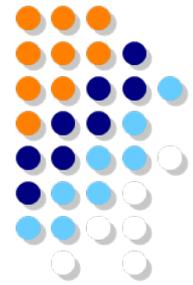


Comparison of Maximum Fluid Temperature between COMSOL Solutions and Analytical Solution for a Constant Flux Tube



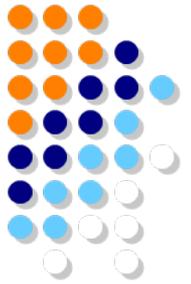
Solution Accuracy (ii)

- Analytic solution within 5% of simulation for all Reynolds numbers
- Model results are conservative
→ Larger maximum temperatures
- Analytic solution provides good basis for initial sizing and feasibility



Conclusions

- New package substrate can enable the use of high temperature coolants
- High thermal conductivity ceramics are necessary to minimize thermal resistance for this coolant path design
- An increase in the nominal flow rate is required to meet the design limitations
- Solution mapping significantly decreases the amount of time required to solve 3-D convective flows
- Work continues to improve flow channel design and thermal conductivity of working fluid



Questions?