

Simulation of Convection in Water Phantom Induced by Periodic Radiation Heating

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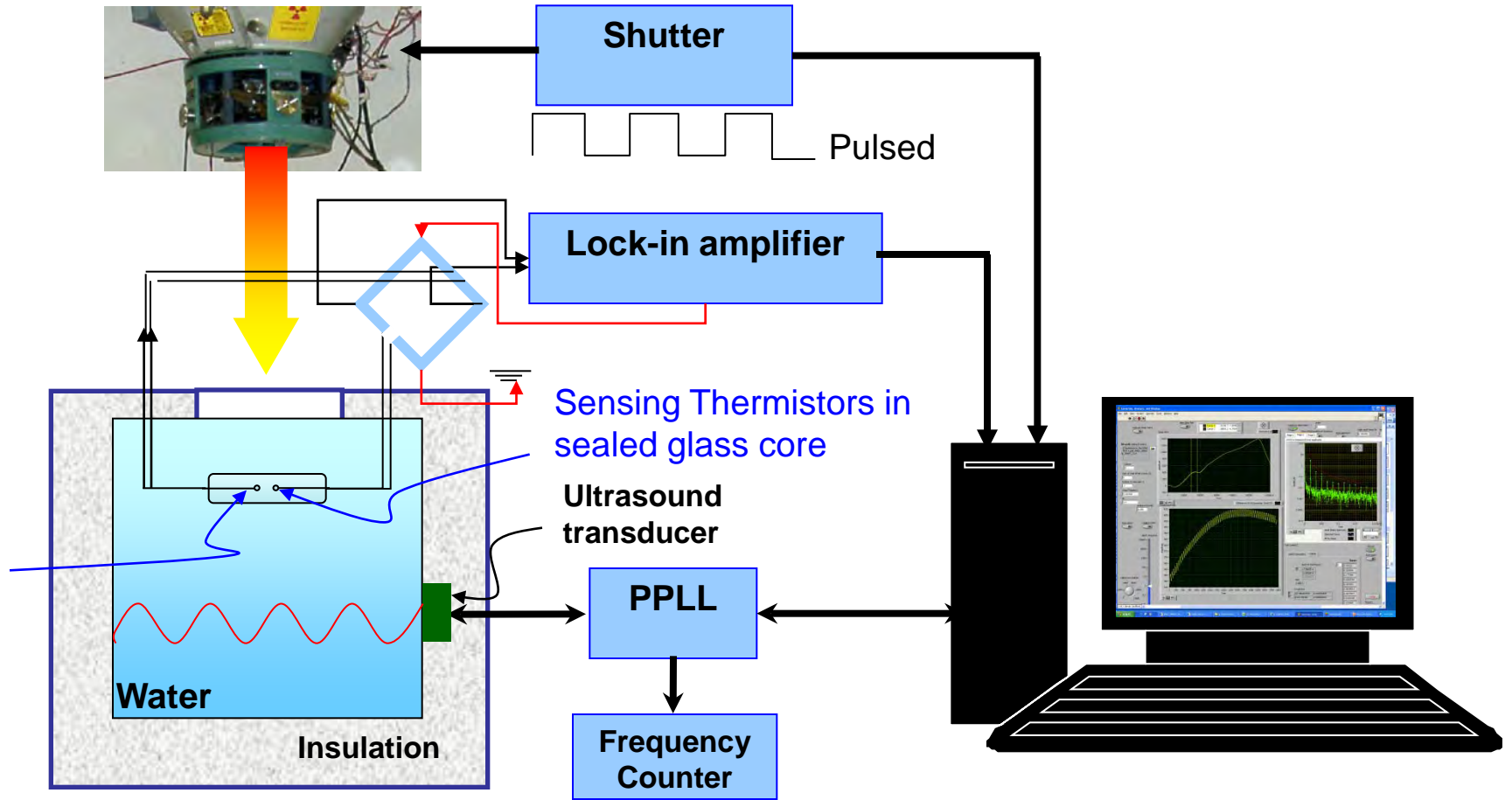
Ionizing Radiation Division

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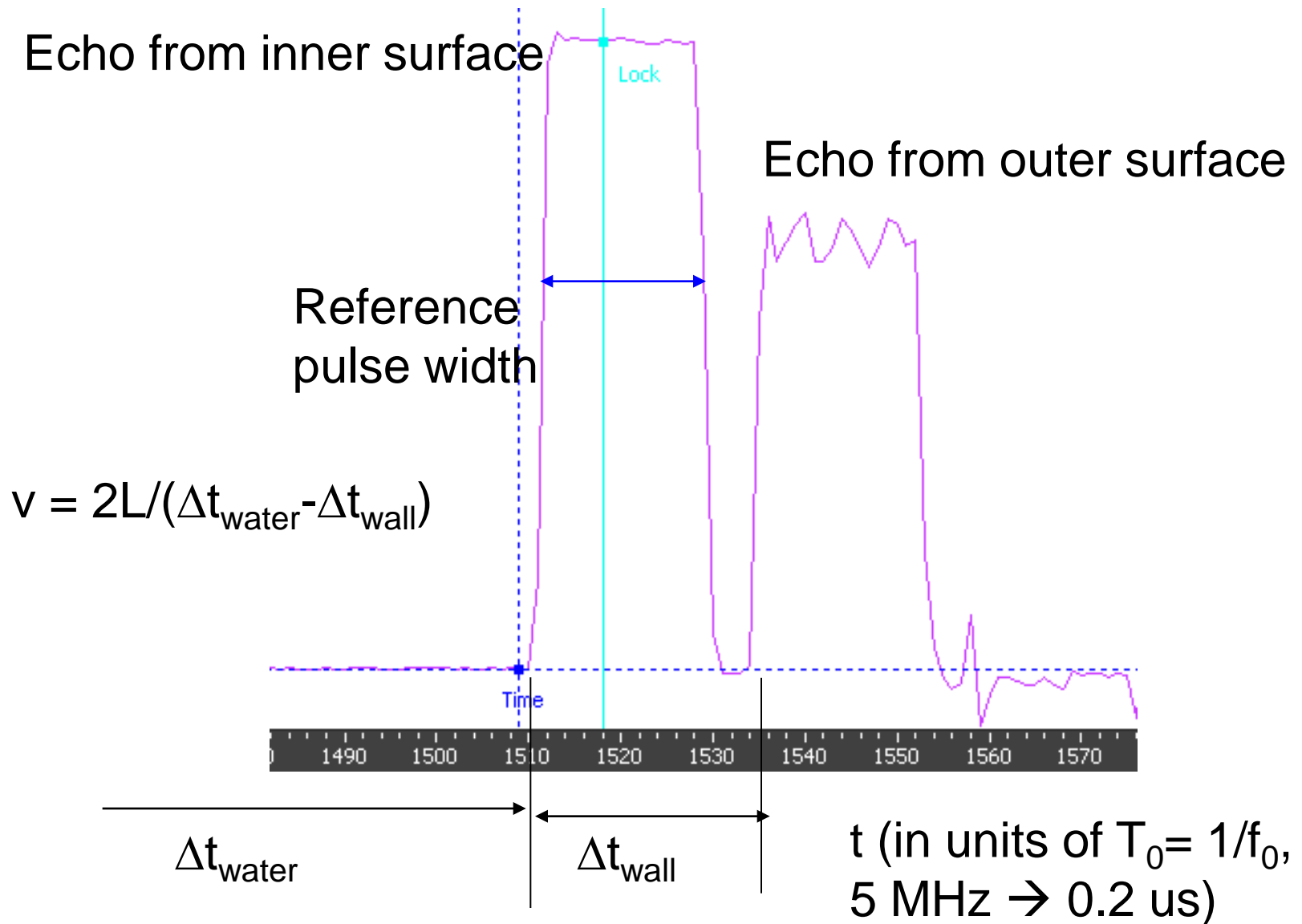
Radiation dosimetry

- Cancer therapy – MeV photons produced from clinical linear accelerator or Co-60
- Ionizing radiation breaks DNA bonds of tumor cells
- Radiation absorbed dose – 1 Gy = 1 J/kg
- $Q \text{ (W/m}^3\text{)} = \rho \text{ (kg/m}^3\text{)} D \text{ (Gy/s)}$
- 1 Gy/min = 17 mGy/s $\rightarrow Q = 17 \text{ W/m}^3$
- Dose: Water calorimetry: in water, 1 Gy causes temperature rise ~ 0.24 mK, yielding a dosimetry primary reference standard
- Dose rate:
1 Gy/min $\rightarrow \Delta T/\Delta t = Q/(\rho C_p) = 17/(1000 \times 4180) = 4 \text{ } \mu\text{K/s}$
- Precise instrument for measuring small temperature changes – sensitive thermistors wired in a Wheatstone bridge with a lock-in amplifier
- Alternative technique – ultrasonic pulsed phase lock loop (PPLL), same sensitivity, non-invasive
- Problem – convection in large phantom caused by temperature gradient due to non-uniform radiation heating

Experimental set up



Ultrasound Thermometer Operation principles



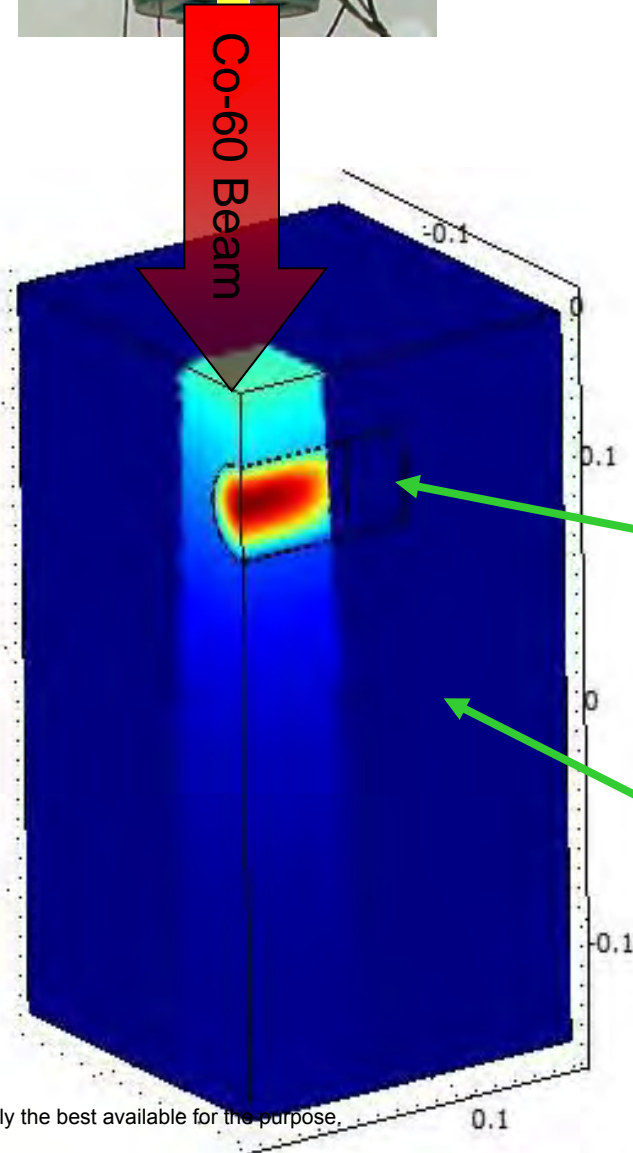
Calibration and noise issues

- Δf , deviation from f_0 , linearly proportional to Δv
- $v = v_0 + a(T-T_0) + b(T-T_0)^2$
- $\Delta T = \alpha(f, T)\Delta f$
- α -- experimentally calibrated $\sim 95.5 \mu\text{K}/\text{Hz}$
- $f_0 = 5 \text{ MHz}$ at $20 \text{ }^\circ\text{C}$
- $v = 1482 \text{ m/s}$
- f can be resolved to 0.01 Hz or better, $\Delta T \sim \mu\text{K}$.
- A root-mean-square (RMS) noise of $3.2 \mu\text{K}$ $30 \times 30 \times 30 \text{ cm}^3$ water tank
- averaging over 400 s
- sampling rate of ~ 1 per second

Typical operation parameters

f_0 Ultrasound detection frequency	5×10^6	Hz
T_0 Ultrasound period	0.2	μs
L Nominal length of ultrasound path	30	cm
Δt Time delay for a round trip in the water tank	240	μs
Reference pulse rate	6000	Hz
Reference pulse width (number of cycles at T_0 period)	18	
Number of "sample and hold" cycles	13	

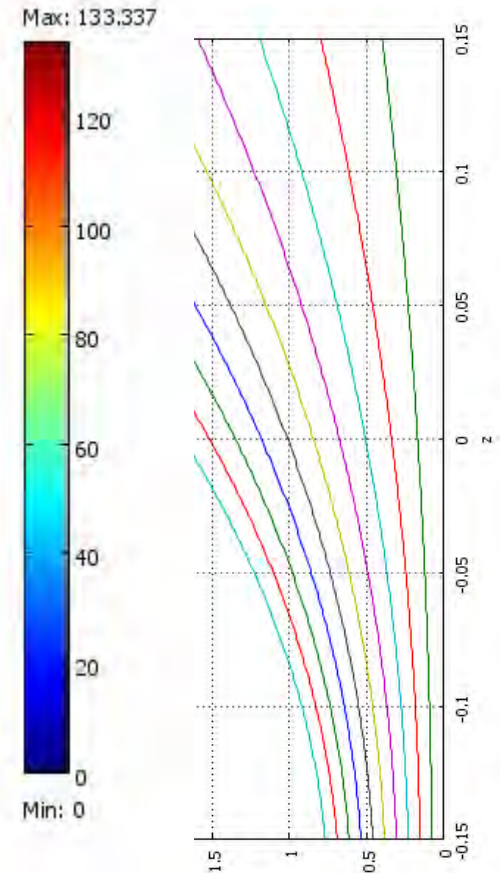
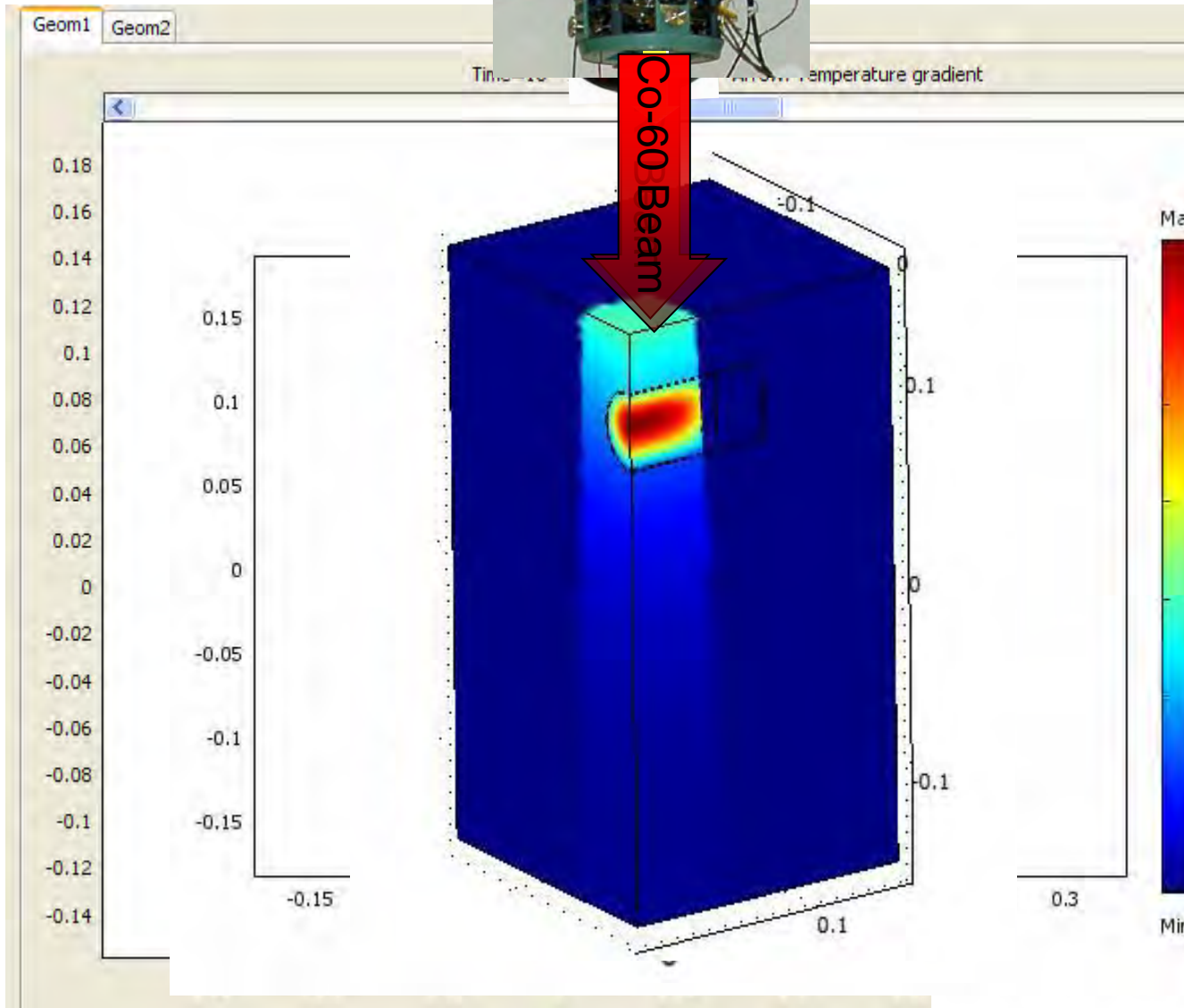
“standard” thermistor-based water calorimetry



Convection less of a problem in thermistor-vessel based water calorimeters, though other labs have gone to 4 C operation to shut off convection altogether.

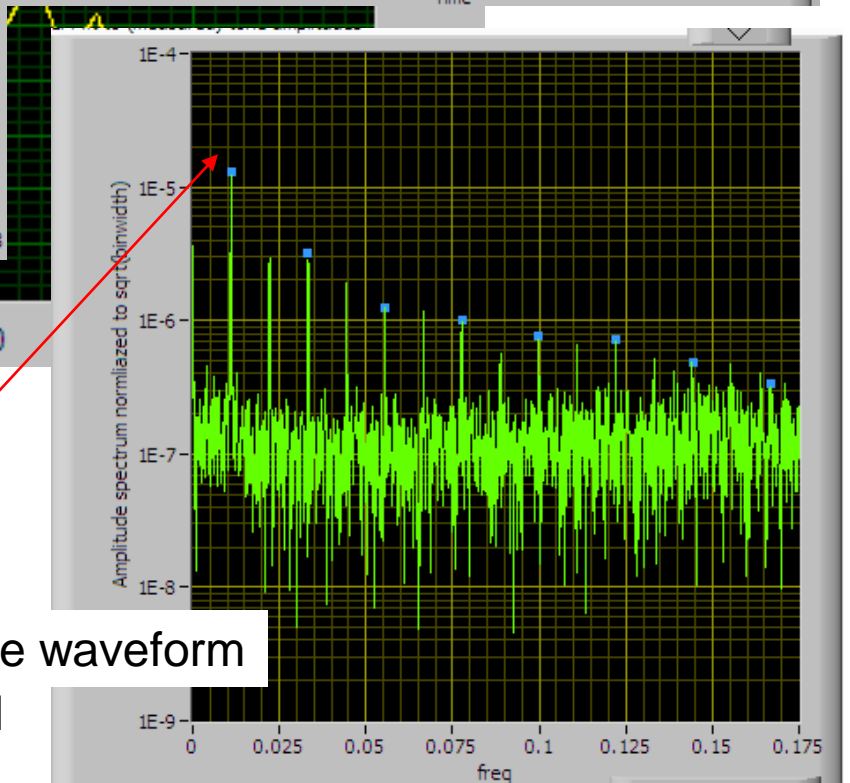
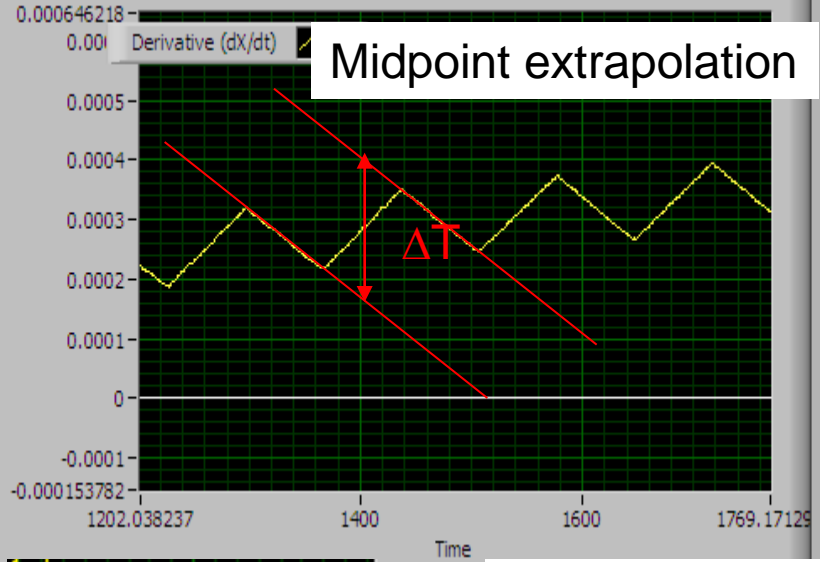
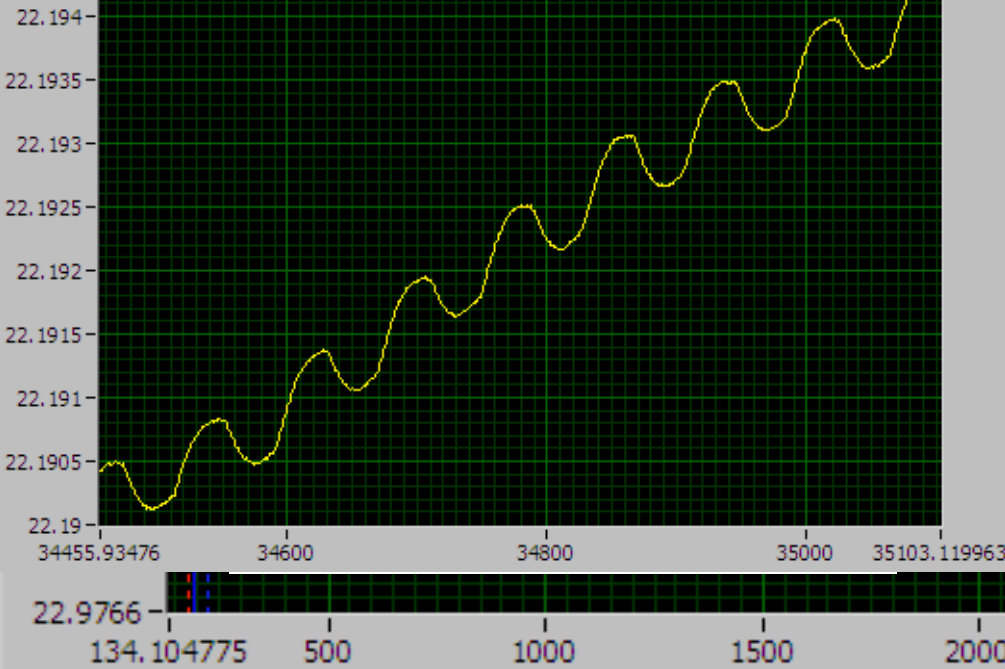
Glass vessel
Containing high purity water
Convection barrier

Outside water stirred



22.9781

Ultrasound temperature waveform
45 s on/off



Tone amplitude $A(f_n)$

Fourier transform of the derivative of the time waveform

$$\Delta T = (\pi/2)A(f_n)n*t_open \quad \text{-- } n \text{ odd}$$

identified are necessarily the best available for the purpose.

Subdomain Settings - Incompressible Navier-Stokes (ns)

Equations

$$\rho \partial_t \mathbf{u} + \rho (\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \eta (\nabla \mathbf{u} + (\nabla \mathbf{u})^T)] + \mathbf{f}$$

$$\nabla \cdot \mathbf{u} = 0$$

$$\partial_t \vec{u} - \nu \nabla^2 \vec{u} = -\nabla \left(\frac{\delta p}{\rho_o} \right) + g \alpha T \hat{z}$$

Subdomains Groups

Subdomain selection

1 2 3 4 5

Physics Stabilization

Fluid properties:

Library material: **Water, liquid** Load...

Subdomain Settings - General Heat Transfer (htgh)

Equation

$$\rho C_p \partial T / \partial t + \nabla \cdot (-k \nabla T) = Q + q_s T - \rho C_p \mathbf{u} \cdot \nabla T + \eta [\nabla \mathbf{u} + (\nabla \mathbf{u})^T - (2/3)(\nabla \cdot \mathbf{u}) \mathbf{I}] : \nabla \mathbf{u} - (T/\rho) \text{div}(\sigma_{ij}) + \mathbf{u} \cdot \nabla p_a$$

T = temperature, Enthalpy H = C_pT/γ + p_a/ρ

$$\partial_t T - \kappa \nabla^2 T = \frac{\dot{D}}{c} g(\vec{x}) f(t) + \vec{u} \cdot \beta \hat{z}$$

Subdomains Groups

Subdomain selection

1 (default) 2 (default) 3 (default) 4 (default) 5 (default)

Group: default

Select by group

Active in this domain

General Convection Ideal Gas Infinite Elements Init Element Stabilization Color

Thermal properties and heat sources/sinks

Library material: **Water, liquid** Load...

Quantity	Value/Expression	Unit	Description
k	k(T)	W/(m·K)	Thermal conductivity
ρ	rho(T)	kg/m ³	Density
C _p	Cp(T)	J/(kg·K)	Heat capacity at constant pressure
q _s	0	W/(m ³ ·K)	Production/absorption coefficient
Q	Q77IR_vbw/2	W/m ³	Heat source
Opacity:	Opaque		

N/m³ Volume force, z dir.

Cancel Apply Help

OK Cancel Apply Help

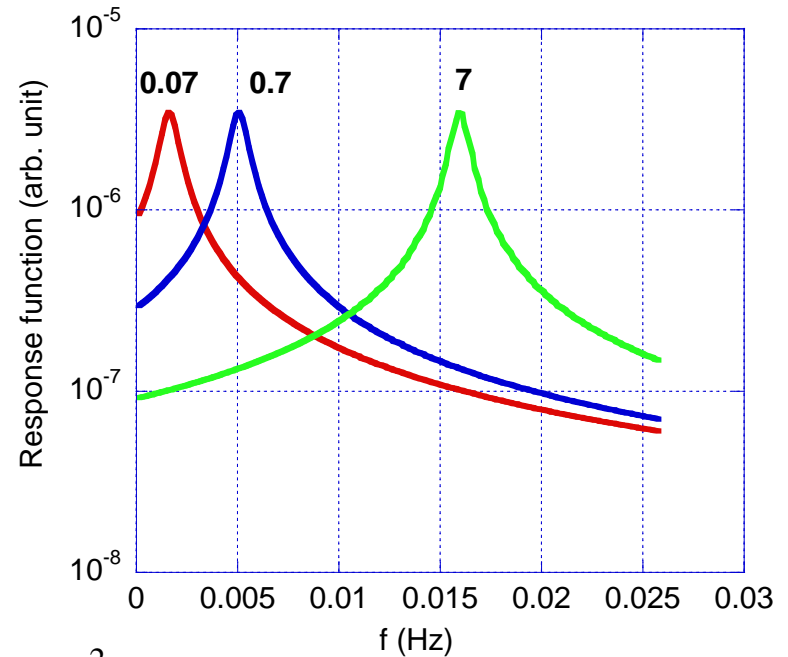
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Response function of a system with a natural oscillation frequency f_0

Lorentzian function

$$f(x; x_0, \gamma) = \frac{1}{\pi\gamma \left[1 + \left(\frac{x-x_0}{\gamma} \right)^2 \right]}$$

$$= \frac{1}{\pi} \left[\frac{\gamma}{(x-x_0)^2 + \gamma^2} \right]$$



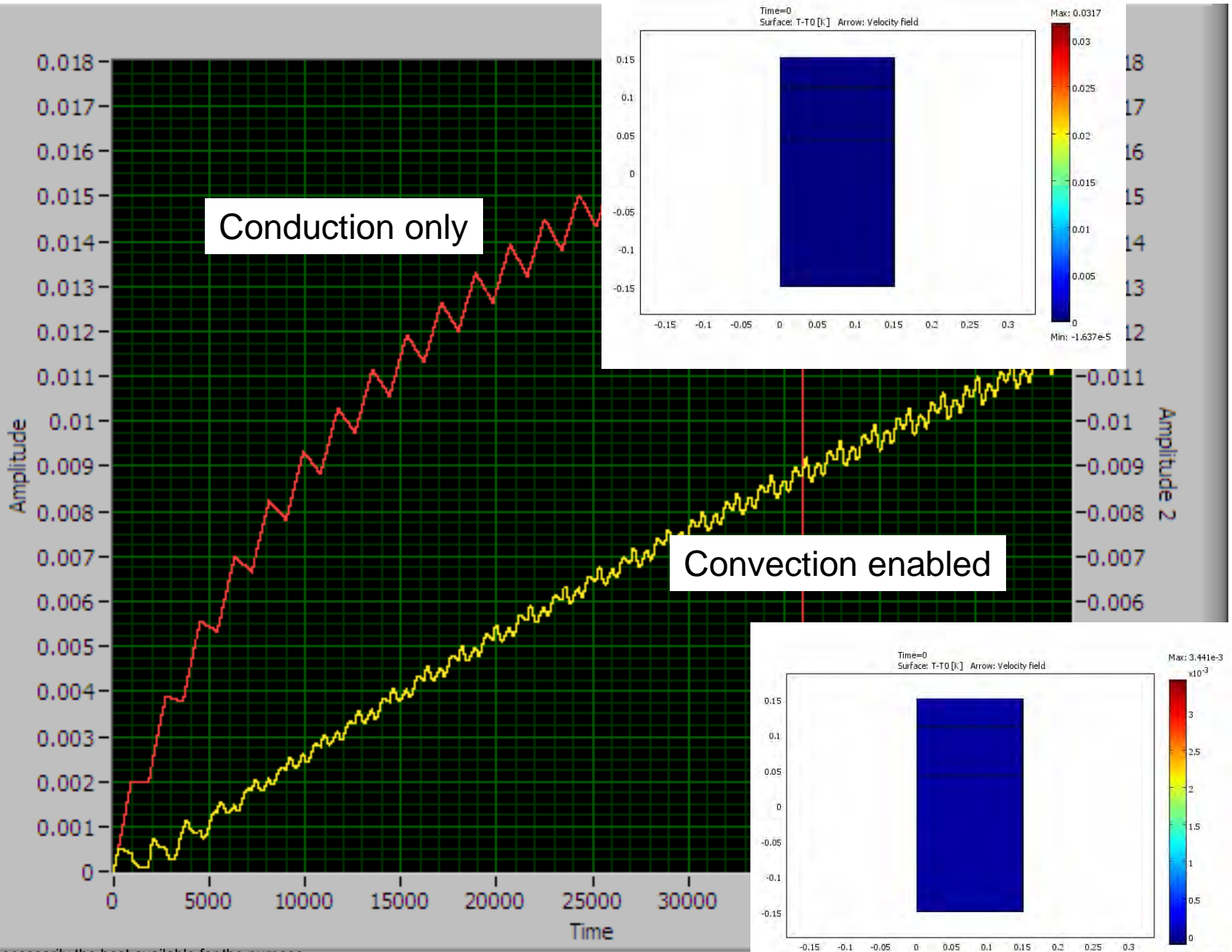
Temperature gradient (K/m)

$$H_{cc}(\vec{x}, \omega) = \sum_{l,m,n=-\infty}^{\infty} \frac{(i\omega + \nu k_u^2 \sigma_{lmn})}{\left(\left[\frac{\beta\alpha g}{\sigma_{lmn}} (l^2 + m^2) + \kappa\nu k_u^2 k_\theta^2 \sigma_{lmn}^2 \right] - \omega^2 \right) + i\omega(\kappa k_\theta^2 + \nu k_u^2) \sigma_{lmn}} g_{lmn}(\vec{x})$$

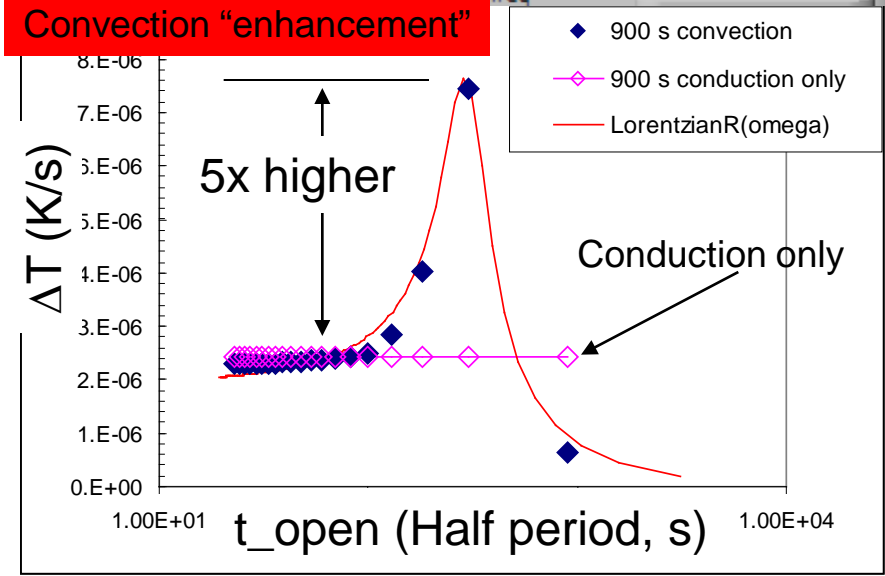
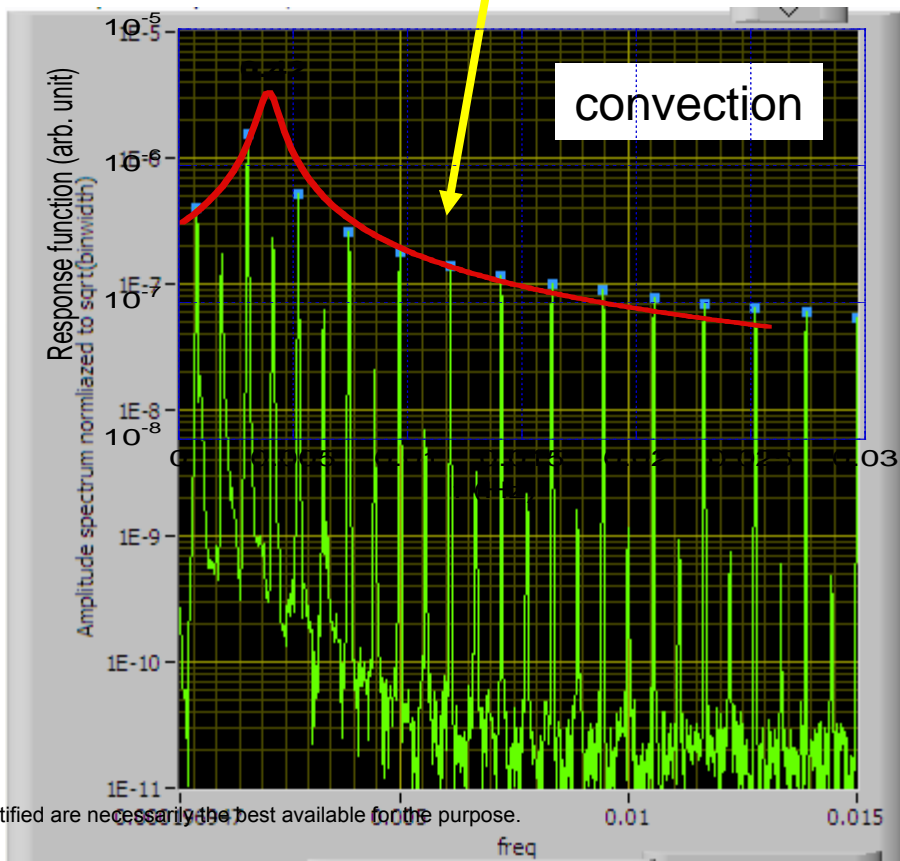
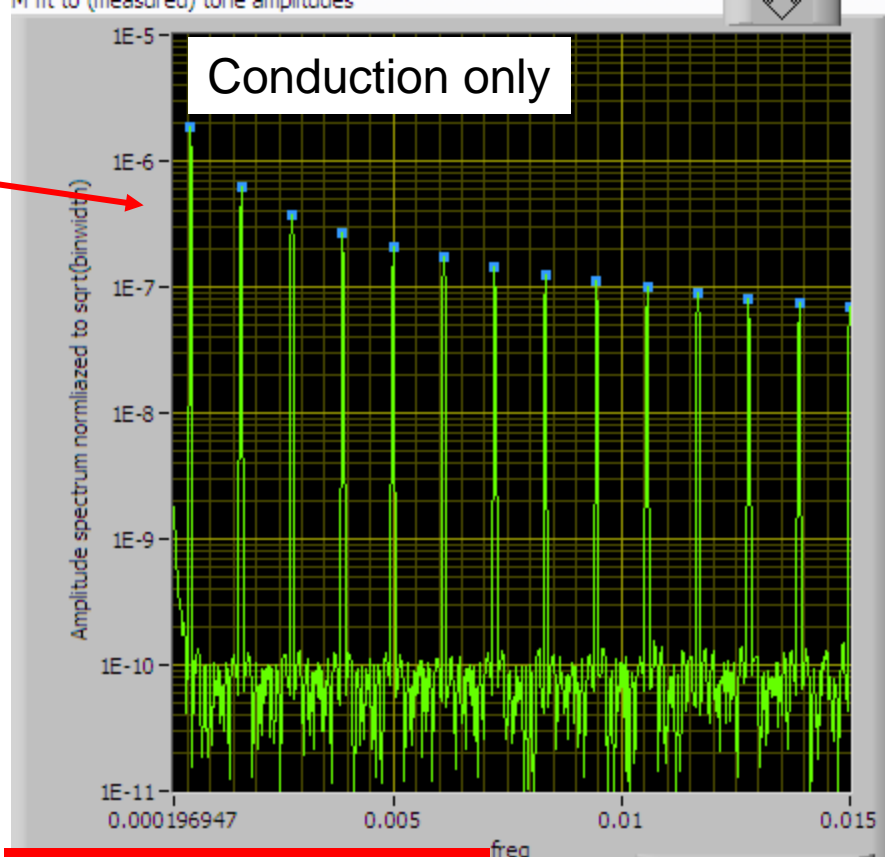
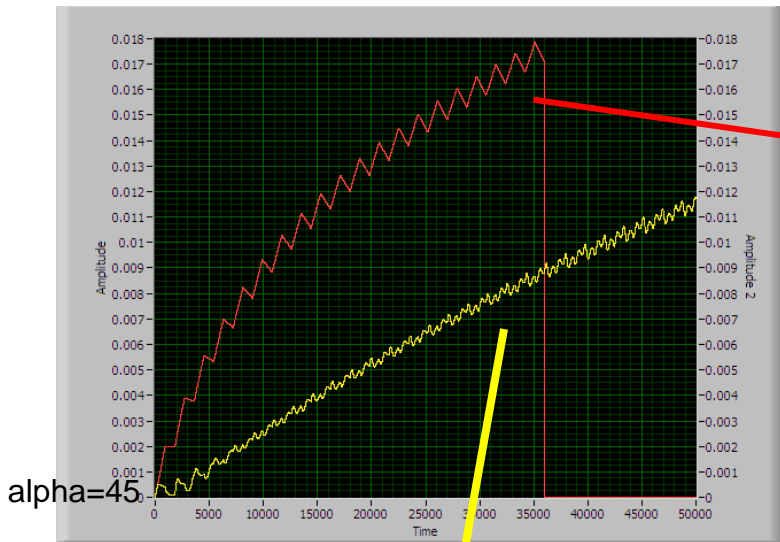
α volume expansion coefficient

ν kinematic viscosity

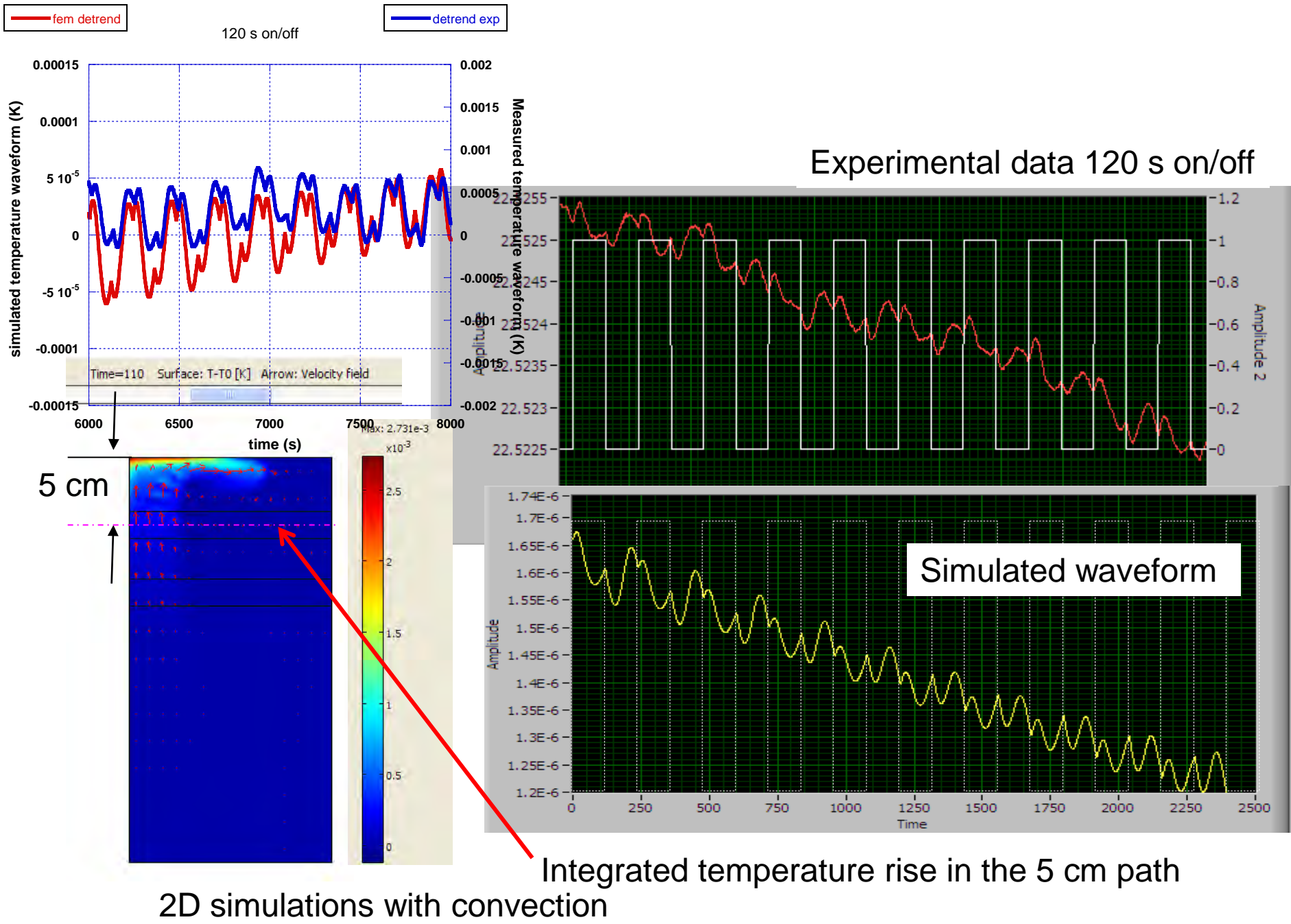
K_θ, k_ν , spatial modes of *temperature and velocity*



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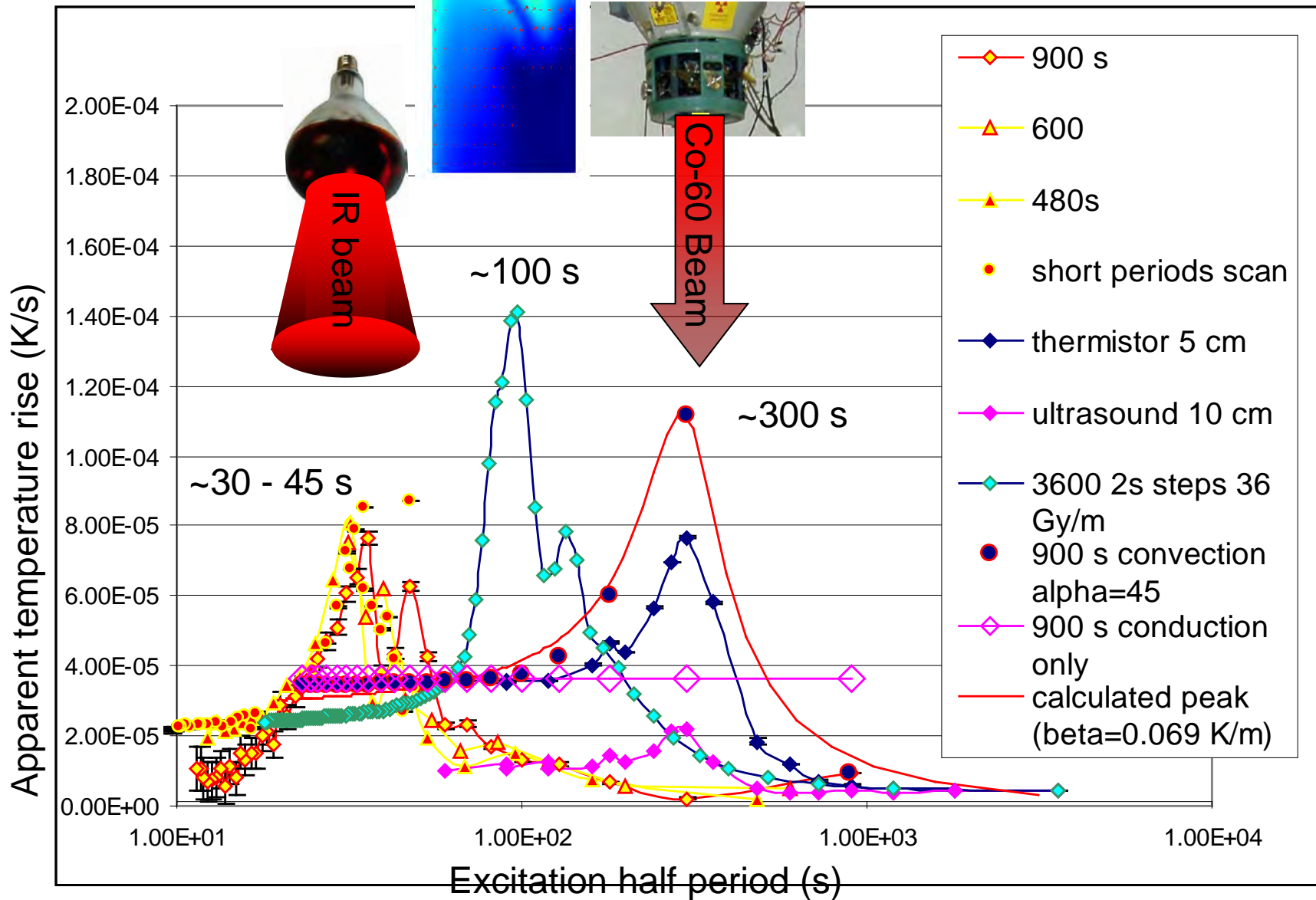


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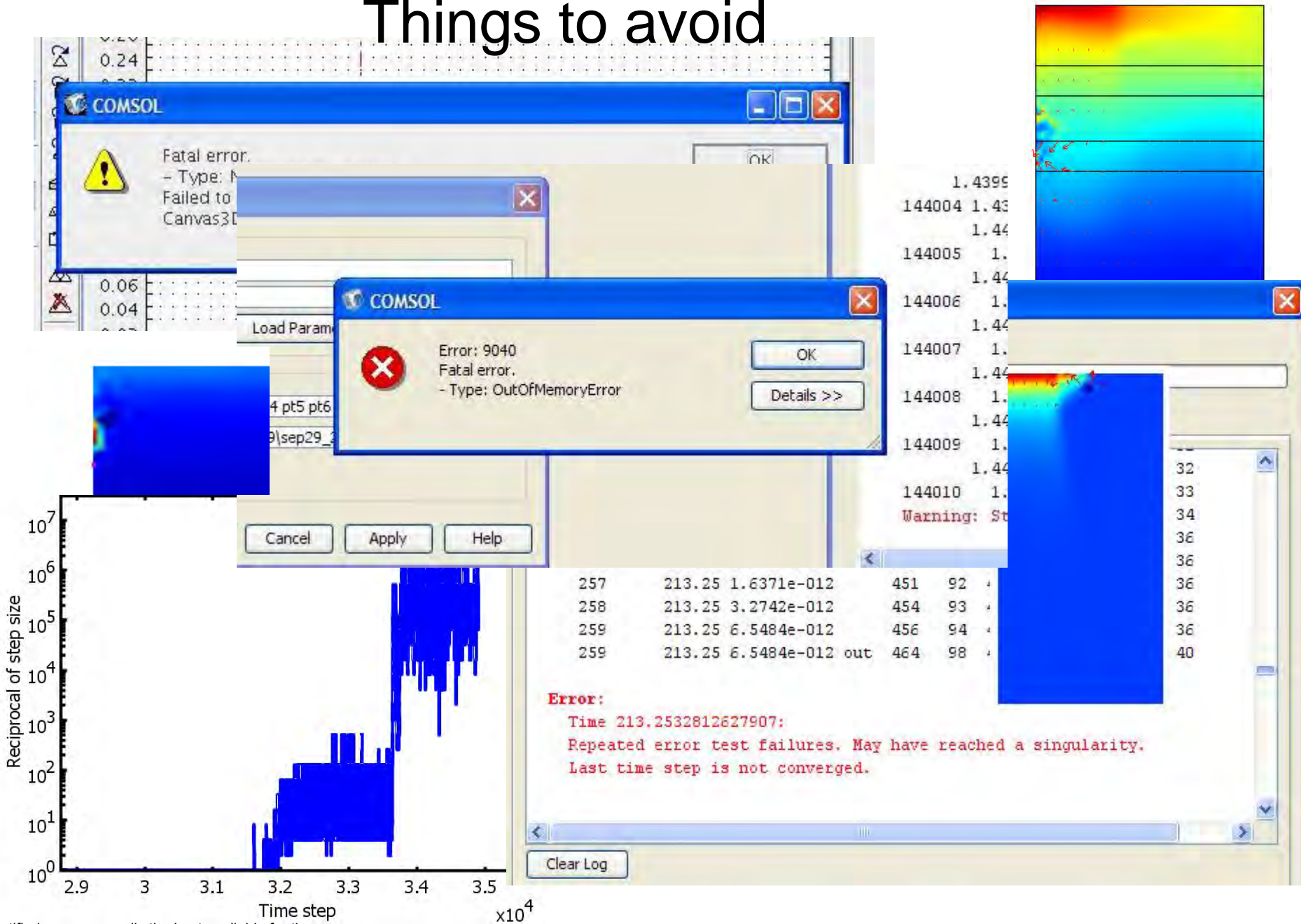
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Summary



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Things to avoid



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Poster Section

Linear Convection and Conduction in Cylinders of Water Exposed to Periodic Thermal Stimuli

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